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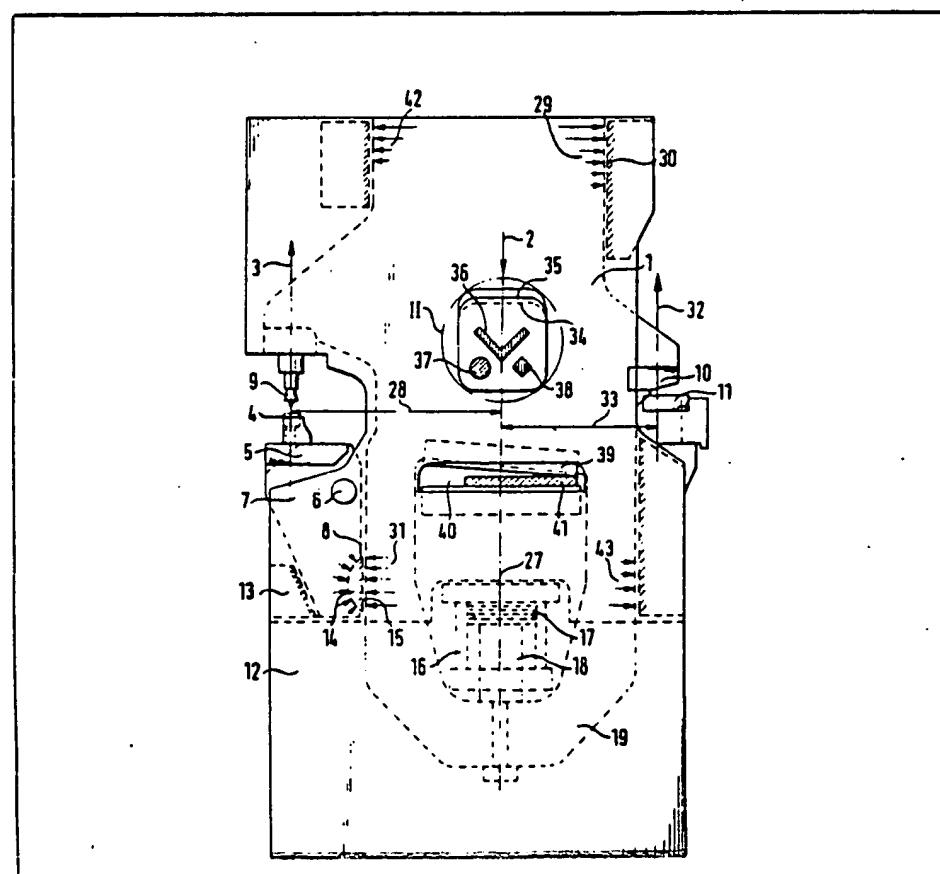
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## (54) Device for cutting or stamping metal bars and sections

(57) This invention relates to a device for cutting or stamping metal bars and sections, the device being of the kind comprising a machine frame (12), a slide (1) which is guided for movement in the frame and driven in the direction of its length by means of an hydraulic or pneumatic drive (16,17) which applies a driving force 2 substantially along the longitudinal axis 27 of the slide, a pair of tools (4,9) of which one (4) is connected to the machine frame and the other (9) to the slide, said pair of tools being laterally offset relatively to the axis 27 and producing a cutting reaction force in the direction of arrow 31 and laterally spaced guides both above and below the tools for guiding said movement of the slide. The

invention provides means (5,7,8) which apply an additional force to the slide (1) to oppose the torque (see arrows 29,31) resulting from the offset location of the tools 4,9 in the form of a saddle 7 pivoted about an axis 6 by the action of punch 9 on the die 4 which is mounted on the saddle, or a second pair of tools 10,11 offset to the opposite side of the axis 27, to equalize wear on the guides. The bearing face 15 of the saddle may be on a partspherical segment 14 self adjustable to lie flush with the slide surface.



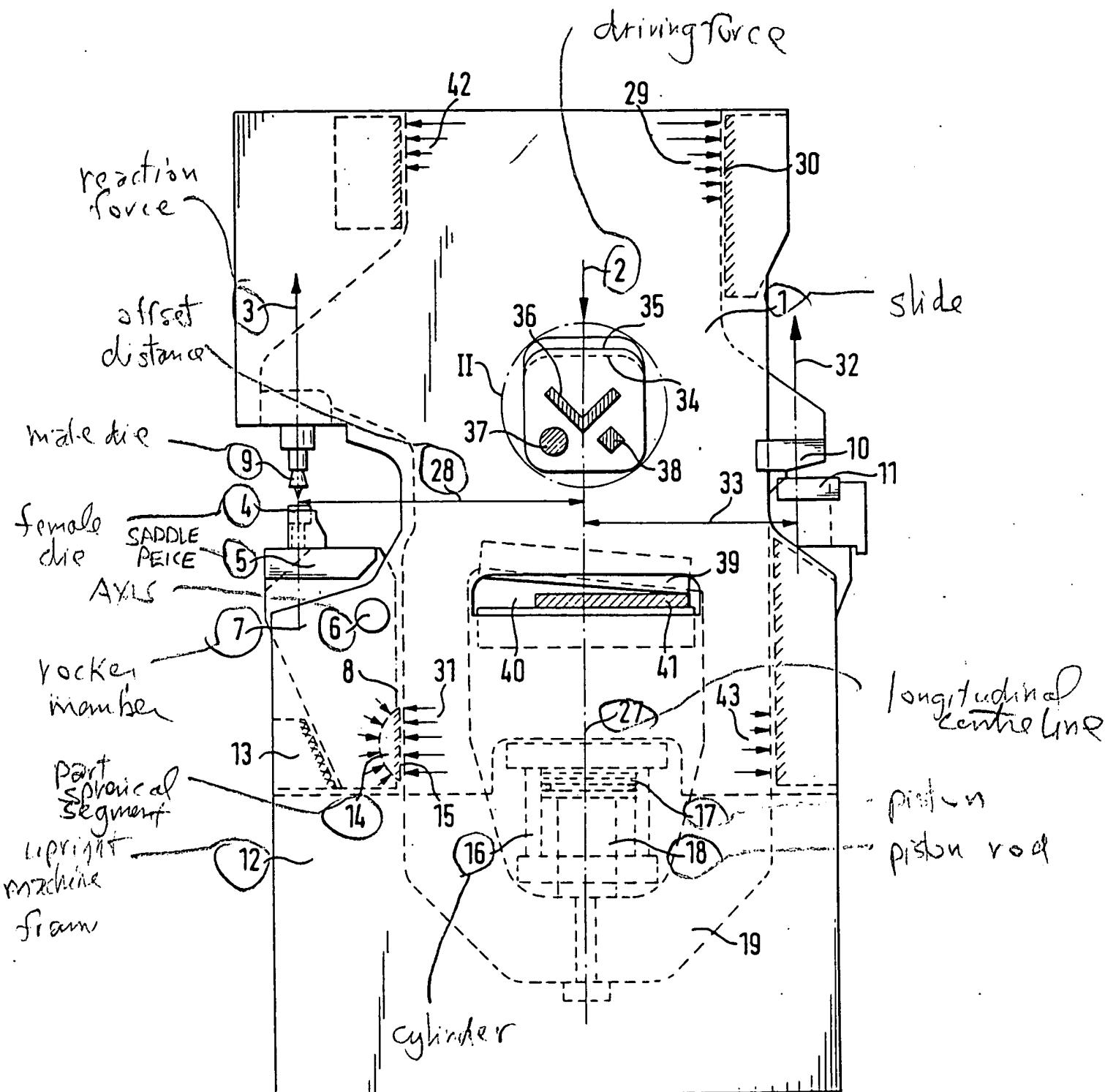


FIG. 1

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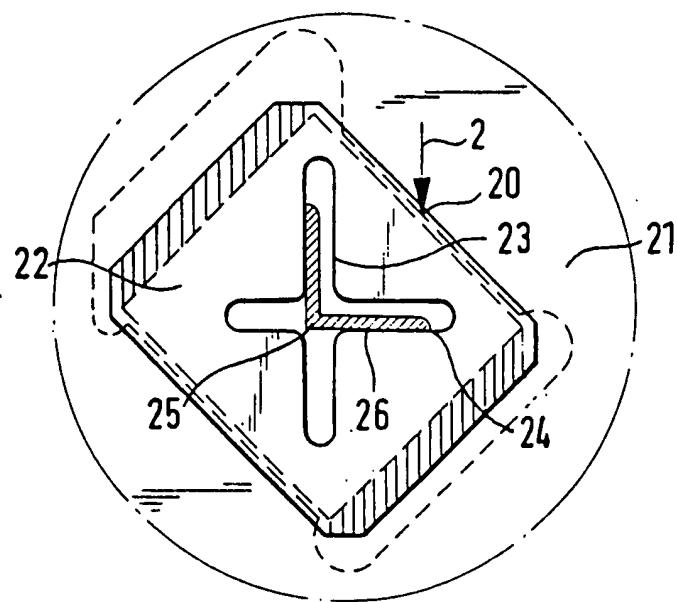


FIG. 2

**SPECIFICATION****Device for cutting and/or stamping metal bars and sections**

5      The invention relates to a device for cutting and/or stamping metal (e.g. steel) bars and sections, said device being of the kind comprising a machine frame adapted to support  
 10     one tool or a pair of tools with a slide guided therein which slide is driven in the direction of its length and is adapted to carry the other tool of said pair of tools, which pair of tools is laterally offset relative to the effective line of  
 15     application to the driving force, laterally spaced guides being provided on the machine frame above and below the tools for guiding the movement of the slide and an hydraulic or pneumatic drive for the slide being provided  
 20     to act substantially in the longitudinal centre line of the slide.

A device of this kind is known from German AS 25 33 153. Owing to the considerable distance between the lines of effective application 25 of the driving force and the cutting reaction force, the guideways are subjected to considerable stress when the device is in operation which entails a correspondingly high degree of wear. Eventually, due to such  
 30     heavy wear, the cuts, or stampings can no longer be sufficiently precisely executed and expensive re-dressing work is required on the device. Wear nearly always leads to a one-sided position of the slide due to the force-couple of driving force and cutting reaction force, and to the distance by which these forces are spaced apart, resulting in wear at diagonally opposite guide regions.

It is the object of the present invention to 40 further develop a device of the kind specified in the sense of enabling it to be operated for substantially longer periods of service without having to incur the added expense of regrinding or replacing the guides. The invention  
 45     stems from the observation that in those regions which are not subjected to stress loads by the one-sided tilt of the slide the respectively affected guide faces of tool slide and machine frame suffer comparatively little  
 50     wear.

According to the present invention, there is 55 provided a device of the kind specified wherein there is provided means which apply an additional force to the slide to oppose the torque moment which is generated by the force couple formed by the driving force and the cutting or stamping reaction force developed in use by said pair of tools.

In this way it is possible either very sub- 60 stantially to reduce guide clearance directly

In a particularly attractive embodiment of the invention the said additional means consist of a saddle part which carries one of the tools and is itself mounted on a rocker member, the latter being hinged to the machine frame for pivotal movement about an axis which intersects the plane of the force couple and having one of its sides adapted to act as a guide face to the slide. By means of this 70 rocker member the actual cutting or stamping force itself is used to oppose the tilting tendency of the slide in as much as the guide face or the rocker member lies against such a part of the slide that in use the rocker member will tend to push the slide in the opposite direction to that direction in which the slide would tend to tilt under the effect of the cutting or stamping force itself. In other words, the rocker member effectively applies a 75 powerful force to the slide only and exclusively during the actual cutting or stamping process.

According to another proposal of the invention the additional means consists of a further 80 set of tools which are arranged on that side to the effective driving force line which is opposite the original set of tools. The concept of this arrangement is based on the consideration that a device which is equipped with two 85 sets of tools arranged one on either side of the effective driving force application line, is also liable to be used for work which involves the actual use of both set of tools. This means that in the course of longer periods of machine service all guide-face areas or regions 90 suffer practically the same amount of wear so that the total service life of the machine is correspondingly prolonged. For example, if identical working tools are provided on both 95 sides of the slide, the equal application, or use, of both sets of tools will eventually afford a considerably longer wear-free operation period than would for example correspond to double the service life of a single set of tools  
 100     provided on one side of the slide only. In this context it is important to remember that wear processes increase progressively with time.

In both embodiments of the invention it is an advantage if the hydraulic or pneumatic 105 drive of the slide is effectively applied in the lower part of the device because this affords substantially improved guide precision corresponding to decreased tilting tendency.

Whereas the rocker member actively influences the guideways when a load is applied to the associated set of tools as hereinbefore described, it also tends to 'give', or yield in the case of a further load application in which further tools are actuated which could cause 110 the slide to cant, or tilt towards the rocker

application line and therefore liable to set up a torque component. In that event, in order to avoid undesired deflection of the rocker member, it is an advantage to limit rocker move-

5 ment by the provision of an end stop on that side thereof which is opposite the guide face. Such an end stop may also quite easily be constructed as an adjustable end stop thereby providing a precision adjustment facility for

10 the device.

A particularly even guide-pressure, or support for the slide can be obtained if a part-spherical segment is mounted in that side of the rocker member which is directed towards

15 the slide and the plane side of such a segment is arranged directly adjacent and abutting the slide. Such a part-spherical segment will always adjust itself flush with the flat surface to be guided thereby and therefore

20 entail a lower specific area compression load.

If the driving force is generated by means of a cylinder secured to the machine frame and the associated extendable piston rod applies a downwardly directed force to the slide

25 for the working stroke thereof, a particularly high operative stroke force can be obtained because in such an arrangement the pressure medium can be applied to the whole of the piston face area in the operative stroke. By

30 contrast, for the return stroke the pressure medium application to the relatively smaller annular area between piston section and piston rod section is quite enough.

The provision of further tools in the vicinity 35 of the longitudinal centre line of the slide is also known from German AS 25 33 153. In order to be able to make precise cuts with the aid of such further tools it is of special advantage to make use of the facility provided by

40 the guide face of the rocker member of the present invention.

To this end the slide is provided with a cutter plate of which the cutting edge is inclined downwardly at an angle of about 45 45°C relative to the line of effective driving force application in a direction away from the rocker member which carries the tool-saddle, the profile-receiving window aperture being provided in the counter-plate. Preferably this 50 aperture will comprise intersecting vertical and horizontal slits for the introduction of angle sections. The horizontal limbs of such angle sections would be pointing away from the rocker member thus allowing take-up of cutting-reaction force on the guide faces opposite the rocker member and its spherical segment. Such an arrangement also preserves the previously known advantage of being able to feed angle sections also obliquely in the horizontal 55 plane for executing mitre-cuts.

For further illustration of this invention refer

tion, and

*Figure 2* is a view representing the area II in Fig. 1 but on a slightly larger scale and in a modified form of execution.

70 Fig. 1 shows a slide 1 which in use is guided, and also driven in the vertical direction, in an upright machine frame 12. Its bottom portion 19 is connected to a piston rod 18 anchored therein, the associated piston 75 ton 17 working in a cylinder 16 under pressure-medium application, the cylinder 16 being mounted on the machine frame 12. When pressure medium is admitted to the upper end of cylinder 16 the slide 1 is displaced in the 80 downward direction, the effective line of application of the driving force 2 coinciding with the longitudinal centre line 27 of the slide 1. The drawing further shows a set of co-acting tools which is laterally offset relative to the 85 effective line of driving force application 2 by distance 28 and comprises a female die, or matrix 4 on a saddlepiece 5 and a male die or punch 9 mounted on a projecting part of the slide 1 above the matrix 4. When a metal 90 (e.g. steel) work-piece is stamped or punched out, a reaction force 3 is generated which, with driving force 2, forms a force-couple whereof the torque moment is determined in part by the value of the said distance 28

95 between the force components. Due to this torque the slide 1 seeks to assume an inclined position bearing on the one side in direction of arrows 29 on the right-hand upper guide face 30 of the machine frame 12 and on the 100 other side in the direction of arrows 31 on the left hand bottom guide face 8. The latter is equipped with a rocker member 7 which is mounted for rocking or pivotal movement about an axis 6. This axis 6 is perpendicular to 105 the plane of the force couple 2,3. The rocker member 7 includes the saddle piece 5 which carries the female die or matrix 4. In order to guarantee a precisely flat area contact between the rocker member 7 and the slide 1,

110 the latter has a part-spherical segment 14 of which the flat or plane side 15 is designed to take up the load at the guide face of slide 1. Fig. 1 further shows another set of tools 10, 11 on the other side of the effective 115 driving force application line 2 consisting of a lower and an upper cutting blade. The lower blade 11 is fitted in the machine frame 12 whereas the upper blade 10 is mounted on the slide 1. Here the cutting stroke sets up a 120 reaction force 32 which, due to the given distance 33 between the line of said reaction force 32 and the line 2 of effective driving force application generates a torque moment pushing the slide in the direction of the upper, 42, and lower, 43 arrows against diag-

side.

Additionally to the tools 4, 9 and 10, 11, the device may be provided with further tools in the region of the longitudinal centre line

5 27. Such further tools may take the form, for example, of a cutter plate 34 secured to the slide 1 and a counter plate 35 secured in the machine frame, the latter plate being provided with suitable passage windows 36, 37, 38 to receive steel or other metal sections and bars of various cross-sectional shapes.

The slide 1 may further carry a shear-blade 39 adapted to be displaced behind a work-receiving window 40 in the machine frame 12 and designed, for example, to cut flat bar sections 41. The width of this window 40 is conveniently equal to double the lateral overhand of the tool set 4, 9 so that flat steel sections of width up to the width of the window can be cut in a single pass and can then be provided by the tool set 4, 9 with stamped or punched holes across the whole of their width in two passes.

Fig. 2 shows (on a larger scale) a particularly attractive modified embodiment of the cutter plate and counter plate unit featured inside the circle II in Fig. 1. As shown in Fig. 2, an angle section 25 is engaged in a vertical-horizontal position in the correspondingly intersecting slits 23, 24 of the work-receiving aperture in a blade-holder plate 22; the horizontal leg 26 must point away from rocker member 7. In region 21 the slide has an edge 20 which extends at an angle of 45° relative to the effective driving force application line 2 and displaces the blade-holder plate 22 at an angle of 45° to said line 2. The said plate has slit openings or windows which correspond configurationally to the workpiece angle section and it is displaceable between the guides of the counter-blade shown hatched in the drawing. In the course of execution of the cutting stroke the cutting force tends to displace the slide to the right-hand side so that it will press against the right-hand upper and lower guide faces without deflecting the rocker member 7. In order to limit the deflection or rocker member 7 during the return stroke and also in order to enable a precise guide adjustment to be obtained, an end stop 13 which is adapted to make plane area contact with the rocker member 7 is provided on the machine frame 12. Any convenient means for adjusting said end stop 13 is provided but is not illustrated in the drawing.

## CLAIMS

1. A device for cutting and/or stamping metal bars and sections of the kind comprising

laterally offset relative to the effective line of application of the driving force, laterally spaced guides being provided on the machine frame above and below the tools for guiding

70 the movement of the slide and an hydraulic or pneumatic drive for the slide being provided to act substantially in the longitudinal centre line of the slide wherein there is provided means which apply an additional force to the

75 slide to oppose the torque moment which is generated by the force couple formed by the driving force and the cutting or stamping reaction force developed in use by said pair of tools.

80 2. A device according to Claim 1, wherein said additional means consists of a saddle which carries said other tool of said pair of tools and which is mounted on a rocker member hinged to the machine frame and having a side which is adapted to serve as a guide face abutting the slide.

85 3. A device according to Claim 1, wherein the additional means comprise a further set of tools which are arranged on the opposite side 90 of the line of application of the driving force to the first set of tools.

95 4. A device according to Claim 1, wherein the hydraulic or pneumatic drive is operatively connected to the slide below said pair of tools.

5. A device according to Claim 2, wherein the deflection of the rocker member is limited by an end stop provided on the machine frame on the opposite side of the rocker member to said guide face thereof.

100 6. A device according to Claim 5, wherein the end stop is adjustable.

7. A device according to Claim 2, wherein a part-spherical segment is inserted in the 105 rocker member so that its plane side abuts the slide and forms said guide face.

8. A device according to Claim 1, wherein the hydraulic or pneumatic drive consists of a cylinder mounted on the machine frame, said 110 cylinder having a piston to which the pressure medium is adapted to be applied and to which is connected a piston rod which extends downwardly out of the cylinder and is connected to the bottom portion of the slide.

115 9. A device according to Claim 2, wherein the slide is provided with an edge in the region of its longitudinal centre line and wherein a blade-holder plate having aperture means designed to receive sections to be sheared is provided in the machine frame,

120 said edge being inclined to the line of application of the driving force at an angle of about 45° in a downwards direction away from said rocker member the section-receiving aperture 125 means in the blade holder plate comprising

metal bars and sections, said device being substantially as hereinbefore described with reference to and as shown in Fig. 1 or Fig. 2 of the accompanying drawing.

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